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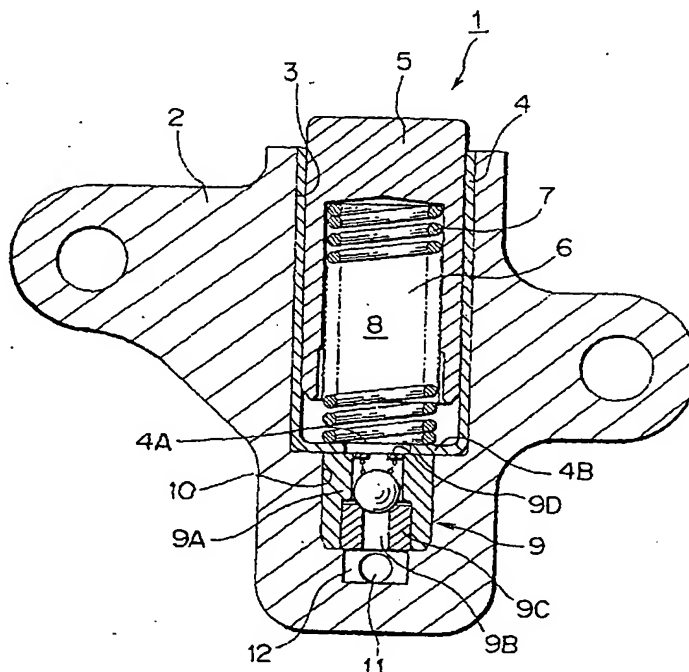
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(54) **Hydraulic tensioner**

(57) A low cost hydraulic tensioner (1) of die-casting metal or synthetic resin with a separate metallic cylinder (4) free from dropping out of the cylinder (4) and fine surface finishing, is provided. The tensioner (1) comprises a metallic cylinder (4) fitted to a cylindrical hole (3) formed in a body (2), a plunger (5) energized by a spring

(7) and a pressure oil chamber (8) formed between the inner part of plunger (5) and the cylinder (4). The cylinder (4) is provided with a bottom plate (4B) having a through hole (4A) allowing the inflow of oil. The bottom plate (4B) is prevented from dropping out from the body (2) by being energized by the spring (7).

FIG. 1

Description

[0001] This invention relates to a hydraulic tensioner for imparting a proper tension to a traveling chain, belt or the like, and particularly to a hydraulic tensioner in which the dropping-out of a metallic cylinder fitted into a cylindrical hole formed in the body of the tensioner is prevented by use of a compression spring provided so as to protrude the tip of a plunger from the body without fixing the cylinder by a separate fixing member or the like.

[0002] Fig. 6 shows a conventional hydraulic tensioner 51 as an example. This hydraulic tensioner 51 comprises an iron-based plunger 54 slidably fitted into a cylinder chamber 53 formed in an iron-made or aluminum alloy-made body 52.

[0003] A hollow part 55 opened at the lower end is formed within the plunger 54, and a compression spring 56 for energizing the plunger 54 to impart a proper tension to a chain, belt or the like (not shown) is housed in the hollow part 55. A pressure oil chamber 57 is formed by the hollow part 55 and the cylinder chamber 53 of the body 52, and a check valve 58 allowing the inflow of oil to the pressure oil chamber 57 but arresting the back flow thereof is provided on the bottom of the pressure oil chamber 57.

[0004] The check valve 58 is formed of a ball seat 59 pressed into a cylindrical hole 63 formed in the lower part of the pressure oil chamber 57, a check ball 61 provided on the upper part of the through-hole 60 of the ball seat 59, a coil spring for energizing the check ball 61 toward the through-hole 60, and a retainer 62 for supporting the end of the coil spring, and the check ball 61 functions as check valve.

[0005] The pressure oil chamber 57 is regularly filled with the oil externally supplied from an inlet port 64 and the through-hole 60 by a pump or the like through the check valve 58. When the traveling chain, belt or the like gets loose, the plunger 54 regularly energized by the compression spring 56 is protruded, and the check valve 58 is opened to allow the oil to flow into the pressure oil chamber 57. When the plunger 54 is pushed into the cylinder chamber 53 by an impact force worked from the chain, belt or the like to the tensioner, the oil pressure in the pressure oil chamber 57 is raised to close the check valve 58.

[0006] When the check valve 58 is closed, the oil in the pressure oil chamber 57 is raised from the lower end of the plunger 54 through a clearance (slight space) between the outer surface of the plunger 54 and the inside surface of the cylinder chamber 53 and discharged to the outside. At this time, the impact energy is absorbed by the flowing resistance of the oil passing through the clearance, and a cushion effect can be obtained.

[0007] However, since the body 52 consists of a die cast product of cast iron or aluminum alloy, the conventional hydraulic tensioner 51 has the problem of requiring film formation by coating, smoothing treatment, ma-

chining or the like to ensure the surface precision or durability of the cylinder chamber for sliding the plunger for the purpose of preventing the wear and seizure of the plunger. Further, it requires an improvement in precision of the cylindrical hole 63 since the ball seat 59 constituting the check valve 58 is pressed into the cylindrical hole 63 formed in the lower part of the pressure oil chamber 57, which causes the problem of requiring the machining therefore.

[0008] When the surface precision and durability of the cylinder chamber is inferior as described above, it is also proposed to clearance-fit a metallic cylinder of separate part into the cylindrical hole formed in the body (Japanese Patent Application Laid-Open No. 2000-346152). In this case, however, it is necessary to form an annular recessed groove on the inside surface of the cylindrical hole and mount a snap ring on the recessed groove for the prevention of dropping-out of the cylinder, resulting in the necessity of processes for recessed groove formation, snap ring mounting and the like, which increases the manufacturing cost by this portion.

[0009] Since the body consisting of the die cast product of cast iron, aluminum alloy or the like is heavy-weight, it is also considered to make the body of plastics in order to reduce the weight. However, the insert molding of a reinforcing part such as metallic cylinder or the like is required because of the presence of problems such as insufficient durability of the sliding surface with the plunger, insufficient strength of each part or the like, causing the problem of complicating a metal mold to increase the cost. Further, a looseness is caused in a temperature rise by the difference in thermal expansion coefficient with a press-in-member such as the ball seat of the check valve, so that high-pressure oil cannot be supplied to the pressure oil chamber.

[0010] In order to solve the problems of the prior arts as described above, this invention has objects to facilitate the mounting of a cylinder and the assembling of a tensioner by dispensing with the film formation, smoothing treatment, machining or the like of a cylinder chamber for sliding a plunger on the body of the tensioner and also dispensing with machining of hole surface or the like in the fitting of the metallic cylinder of separate article, and to prevent the dropping-out of the cylinder by use of a compression spring provided so as to protrude the plunger from the body without requiring any cylinder fastener such as snap ring of separate member. Further, it has objects to prevent also the dropping-out of a check valve mechanism by use of the bottom plate of the cylinder in the mounting of the check valve mechanism and to facilitate the assembling thereof.

[0011] In order to attain the above objects, a hydraulic tensioner according to a first aspect: of this invention comprises a metallic cylinder fitted to a cylindrical hole formed in a body, a plunger provided within the cylinder so as to be energized by a compression spring to protrude the tip from the body, and a pressure oil chamber

formed between the inner part of the plunger and the cylinder, wherein the cylinder is provided with a bottom plate having a through-hole allowing the inflow of oil in the center, and the bottom plate is prevented from dropping out from the body by being energized by the compression spring.

[0012] According to a second aspect of this invention, in the hydraulic tensioner of the first aspect, the cylindrical hole is provided with a cylindrical hole smaller in diameter than this cylindrical hole on the bottom side, and a check valve mechanism fitted to the minor diameter cylindrical hole is prevented from dropping out by the abutment on the bottom plate of the cylinder.

[0013] According to a third aspect of this invention, a hydraulic tensioner comprising a metallic cylinder fitted to a cylindrical hole formed in a body, a plunger provided within the cylinder so as to be energized by a compression spring to protrude the tip from the body, a pressure oil chamber formed between the inner part of the plunger and the cylinder, and a check valve mechanism allowing the inflow of oil to the pressure oil chamber but arresting the back flow thereof, wherein the cylinder is provided with a bottom plate having a through hole allowing the inflow of oil in the center, and the bottom plate being prevented from dropping out from the body by being energized by the compression spring, the check valve mechanism comprises a check ball provided so as to block an inflow hole of oil to the pressure oil chamber from above and a spring for energizing the check ball and the spring is supported by the bottom plate, is provided.

[0014] Further, according to a fourth aspect of this invention, a hydraulic tensioner according to the third aspect, wherein the check valve mechanism further comprises a lid for supporting the spring and being a preliminary assembled product, and the lid is supported by the bottom plate, is provided.

[0015] A hydraulic tensioner according to a fifth aspect of this invention comprises a metallic cylinder fitted to a cylindrical hole formed in a body, a plunger provided within the cylinder so as to be energized by a compression spring to protrude the tip from the body, a pressure oil chamber formed between the inner part of the plunger and the cylinder, and a check valve mechanism allowing the inflow of oil to the pressure oil chamber but arresting the back flow thereof, wherein the cylinder is provided with a bottom plate having a through-hole allowing the inflow of oil in the center, the bottom plate is prevented from dropping out from the body by being energized by the compression spring through the retainer of the check valve mechanism, and the check valve mechanism comprises a check ball provided so as to block the through-hole formed in the bottom plate from above, a spring for energizing the check ball, and the retainer for supporting the spring.

[0016] According to a sixth aspect of this invention, in a hydraulic tensioner according to any one of the preceding aspects, a ratchet pawl body to be engaged with

a ratchet tooth engraved in the outside surface of the plunger is rockably provided on the body by spring energization in order to prevent the backward displacement of the plunger, a cutout for rocking the ratchet pawl body engageably with the ratchet tooth and a pair of protruding pieces extending outward from the end of the cutout are formed on the cylinder, and the ratchet pawl body is supported through a shaft inserted to mount holes formed in the protruding pieces and a mount hole formed in the body.

[0017] According to a seventh aspect of this invention, a hydraulic tensioner according to any one of the preceding aspects, wherein the body is made by die-casting and the cylinder is made of steel iron, is provided.

[0018] Moreover, according to an eighth aspect of this invention, a hydraulic tensioner according to any one of the preceding aspects, wherein the body is made by synthetic resin molding and the cylinder is made of steel iron.

[0019] In this invention, the body may consist of a die cast product of cast iron, aluminum, aluminum alloy, a plastic product or the like without specially limiting the material thereof. As the cylinder, any metallic cylinder of steel iron, cast iron, sintered metal or the like can be used if it never causes a seizure with the plunger without specially limiting the material thereof.

[0020] When the hydraulic tensioner of this invention is mounted on an engine, a drive unit or the like, the compression spring energizes the bottom plate of the cylinder since the plunger provided so as to protrude the tip from the body of the tensioner is pressed by a chain, belt or the like and pushed so as to retreat into the cylinder, and the dropping-out of the cylinder can be thus prevented without separately providing any fastener. The thus-prevention of dropping-out of the bottom plate of the cylinder by the compression spring allows the clearance fit of the cylinder. When the die cast product of cast iron, aluminum, aluminum alloy or the like or the plastic product by synthetic resin molding is used as the body, the cylindrical hole for fitting the cylinder requires no surface treatment, machining or the like, and the surface treatment, machining or the like of the mounting surface to the engine, drive unit or the like is also dispensed with in the use of the plastic product.

Fig. 1 is a sectional view of a tensioner according to Example 1 of this invention.

Fig. 2 is a sectional view of a tensioner according to Example 2 of this invention.

Fig. 3 is a plan view of a tensioner according to Example 3 of this invention.

Fig. 4 is a sectional view of the tensioner of Fig. 3.

Fig. 5 is a perspective view of the tensioner of Fig. 3.

Fig. 6 is a sectional view of a conventional tensioner.

[0021] A preferred embodiment of this invention is described on the basis of Example 1 in reference to Fig. 1. Fig. 1 shows a section of a hydraulic tensioner 1, and

the tensioner 1 comprises a steel iron-made cylinder 4 fitted to a cylindrical hole 3 formed in a body 2 and a plunger 5 slidably inserted into the cylinder 4. In this case, the cylinder may be clearance-fitted. The cylindrical hole 3 is provided with a cylindrical hole 10 smaller in diameter than this cylindrical hole 3 on the bottom side. The cylinder 4 is provided with a bottom plate 4B having a through-hole 4A communicating with a check valve mechanism 9 described later in the center.

[0022] A hollow part 6 opened at the lower end is formed within the plunger 5, and a compression spring 7 for protruding the tip of the plunger 5 to impart a proper tension to a chain, belt or the like is housed in the hollow part 6. A pressure oil chamber 8 is formed by the hollow part 6 and the inner part of the cylinder 4, and the check valve mechanism 9 allowing the inflow of oil to the pressure oil chamber 8 but arresting the back flow thereof is provided on the bottom of the pressure oil chamber 8.

[0023] The check valve mechanism 9 is a preliminarily assembled product (so-called assembly) consisting of a cylindrical block 9A, a ball seat 9C having an oil passage 9B, a check ball, a coil spring, and a lid 9D. It is fitted to the minor-diameter cylindrical hole 10 provided in the bottom side of the cylindrical hole 3 and prevented from dropping out by the abutment of its upper part on the bottom plate 4A of the cylinder 4.

[0024] The pressure oil chamber 8 is regularly filled with the oil externally supplied from an oil inlet port 11, an oil chamber 12 and the oil passage 9B by a pump or the like through the check valve mechanism 9. When the traveling chain, belt or the like gets loose, the plunger 5 regularly energized by the compression spring 7 is protruded, and the check valve mechanism 9 is also opened to allow the oil to flow into the pressure oil chamber 8. When the plunger 5 is pushed into the cylinder 4 by an impact force worked from the chain, belt or the like to the tensioner, the oil pressure in the pressure oil chamber 8 is raised to close the check valve mechanism 9.

[0025] When the thus-formed hydraulic tensioner 1 is mounted on an engine, a drive unit or the like, the compression spring 7 regularly energizes the bottom plate 4A of the cylinder 4 since the plunger 5 provided so as to protrude the tip from the body 2 of the tensioner is pressed by the chain, belt or the like and pushed into the cylinder 4, and the dropping-out of the cylinder 4 can be thus prevented.

[0026] Since the prevention of dropping-out of the cylinder 4 allows the clearance fit of the cylinder 4, and the cylindrical hole 3 for fitting the cylinder 4 requires no surface treatment, machining or the like when the die cast product of cast iron, aluminum, aluminum alloy or the like or the plastic product by synthetic resin molding is used as the body 2. Since the check valve mechanism 9 fitted to the minor-diameter cylindrical hole 10 provided in the bottom side of the cylindrical hole 3 is prevented from dropping out by the abutment on the bottom plate 4A of the cylinder 4, the surface treatment, ma-

chining or the like of the minor-diameter cylindrical hole 10 is also unnecessary. Consequently, the press-in of the cylinder, the ball seat or the like is dispensed with.

[0027] Example 2 is described on the basis of Fig. 2. For each example described below, the same reference number is imparted to the same member as in Example 1. Fig. 2 shows a section of a hydraulic tensioner 21, and the tensioner 21 comprises a steel iron-made cylinder 24 fitted to a cylindrical hole 23 formed in a body 22 and a plunger 25 slidably inserted into the cylinder 24. The cylinder 24 is provided with a bottom plate 24B having a through-hole 24A allowing the inflow of the oil constituting a check valve mechanism 29 in the center as described below.

[0028] A hollow part 26 opened at the lower end is formed within the plunger 25, and a compression spring 27 for protruding the tip of the plunger 25 is housed in the hollow part 26. A pressure oil chamber 28 is formed by the hollow part 26 and the inner part of the cylinder 24, and the check valve mechanism 29 allowing the inflow of oil to the pressure oil chamber 28 but arresting the back flow thereof is provided on the bottom of the pressure oil chamber 28. Denoted at 24C in Fig. 2 is a cylindrical member extended down from the through-hole 24A, which forms an oil passage.

[0029] The check valve mechanism 29 comprises a check ball provided so as to block the through-hole 24A formed in the bottom plate 24B of the cylinder 24 from above, a spring for energizing the check ball, and a retainer 29A for supporting the spring, and it is formed above the bottom plate 24B of the cylinder 24.

[0030] The pressure oil chamber 28 is regularly filled with the oil externally supplied from an oil inlet port 11 and an oil chamber 12 by a pump or the like through the check valve mechanism 29. When the traveling chain, belt or the like gets loose, the plunger 25 regularly energized by the compression spring 27 is protruded, and the check valve mechanism 29 is opened to allow the oil to flow into the pressure oil chamber 28. When the plunger 25 is pushed into the cylinder 24 by an impact force worked from the chain, belt or the like to the tensioner, the oil pressure in the pressure oil chamber 28 is raised to close the check valve mechanism 29.

[0031] When the thus-formed hydraulic tensioner 21 is mounted on an engine, a drive unit or the like, the compression spring 27 regularly energizes the bottom plate 24B of the cylinder 24 through the retainer 29A of the check valve mechanism 29 since the plunger 25 provided so as to protrude the tip from the body 22 of the tensioner is pressed by the chain, belt or the like and pushed into the cylinder 24, and the dropping-out of the cylinder 24 can be thus prevented.

[0032] Example 3 is described on the basis of Figs. 3-5. A hydraulic tensioner 31 of Example 3 consists of the hydraulic tensioner of Example 1 provided with a backward displacement preventing means for plunger in which a ratchet pawl pivotally fixed to the body is engaged with a ratchet tooth engraved in the plunger as

described below in detail. Fig. 4 shows a section of the hydraulic tensioner 31, and the tensioner 31 comprises a steel iron-made cylinder 34 clearance-fitted to a cylindrical hole 33 formed in a plastic body 32, and a plunger 35 slidably inserted into the cylinder 34.

[0033] A hollow part 36 opened at the lower end is formed within the plunger 35, and a compression spring 37 for protruding the tip of the plunger 35 is housed in the hollow part 36. A pressure oil chamber 38 is formed by the hollow part 36 and the inner part of the cylinder 34, and a check valve mechanism 39 allowing the inflow of oil to the pressure oil chamber 38 but arresting the back flow thereof is provided on the bottom of the pressure oil chamber 38.

[0034] The check valve mechanism 39 has the same structure as the check valve mechanism 9 and mounting means as Example 1, and consists of a preliminarily assembled product consisting of a cylindrical block; a ball seat having an oil passage, a check ball, a coil spring and a lid. It is fitted into a cylindrical hole smaller in diameter than the cylindrical hole 33 further formed in the bottom of the cylindrical hole 33.

[0035] A recessed groove 32B communicating with the cylindrical hole 33 is formed in the body 32, and a ratchet pawl body 40 to be engaged with a ratchet tooth 35A engraved in the outside surface of the plunger 35 is rockably provided within the recessed groove while being energized by a ratchet spring 41 in order to prevent the backward movement of the plunger 35. In this case, a backlash is provided between the ratchet tooth 35A and the pawl of the ratchet pawl body 40.

[0036] The cylinder 34 comprises a bottom plate 34B having a through-hole 34A communicating with a check valve mechanism 39 described below in the center. Further, as shown in Figs. 3 and 5, the cylinder 34 comprises a cutout 34C for rocking the ratchet pawl body 40 engageably with the ratchet tooth 35A of the plunger 35, and a pair of protruding pieces 34D and 34D extending outward from the end of the cutout 34C. The ratchet pawl body 40 is inserted to mount holes 34E formed in the protruding pieces 34D and a mount hole 32A formed in the body 32, and supported through a shaft hole 40A by a shaft 42 inserted to the mount hole 32A.

[0037] The action of the hydraulic tensioner 31 is as follows. The pressure oil chamber 38 is regularly filled with the oil externally supplied from the oil inlet port 11, the oil chamber 12 and the oil passage by a pump or the like through the check valve mechanism 39. When the traveling chain, belt or the like gets loose, the plunger 35 regularly energized by the compression spring 37 is protruded, and the check valve mechanism 39 is opened to allow the oil to flow into the pressure oil chamber 38.

[0038] When the plunger 35 is pushed into the cylinder 34 against the energizing force of the compression spring 37 by an impact worked from the chain, belt or the like to the tensioner, the plunger 35 is retreated within the range of the backlash while the ratchet tooth 35A

is slid on the pawl of the ratchet pawl body 40, the oil pressure in the pressure oil chamber 38 is raised to close the check valve mechanism 39, and the impact is moderated. In this case, the engagement of the ratchet tooth 35A with the pawl of the ratchet pawl body 40 prevents the plunger 35 from retreating more than it is required.

[0039] When the chain, belt or the like is elongated by a long-term operation, and the plunger 35 is protruded from the body 32 exceeding the range of the backlash, the check valve mechanism 39 is opened to supply the oil to the pressure oil chamber 38 through the inlet port 11, and the engagement of the ratchet tooth 35A of the plunger 35 with the pawl of the ratchet pawl body 40 is also shifted by one tooth, so that the plunger 35 is regularly moved outward the body 32 following the elongation of the belt or chain.

[0040] In the thus-constituted hydraulic tensioner 31, since the plunger is pressed by the chain, belt or the like and pushed so as to retreat into the cylinder, the bottom plate of the cylinder is energized downward by the compression spring, and protruding pieces 34D formed on the cylinder 34 are also fixed to a mount hole 32A formed in the body 32 through a shaft 42. Accordingly, the dropping-out from the cylindrical hole 33 formed in the body 32 can be prevented.

[0041] Since the shaft 42 for supporting the ratchet pawl body 40 is inserted to the mount holes 34E of the protruding pieces 34D of the steel iron-made cylinder 34, the shaft 42 can be particularly advantageously supported, when the body 32 is made of plastics, aluminum alloy or the like inferior in strength, by the steel iron-made protruding pieces 34D without breaking the mount hole 32A of the body 32. Since the dropping-out of the cylinder 34 is prevented in this way, surface precision or pore diameter precision is not needed for the cylindrical hole for fitting the cylinder, and a process of surface treatment, machining or the like can be omitted to reduce the manufacturing cost.

[0042] The hydraulic tensioner provided with the check valve mechanism is described in Examples 1-3, but the check valve mechanism is not necessarily required. Since the high-pressure oil is supplied from an oil pump to the pressure oil chamber in this case, a cushion effect can be obtained by the oil pressure within the pressure oil chamber when the plunger is pushed into the cylinder by the impact force worked from the chain, belt or the like to the tensioner.

[0043] In this invention, the wear resistance can be improved since the metallic cylinder is fitted to the cylindrical hole formed in the body as described above. Further, since the cylinder is provided with the bottom plate having the through-hole allowing the inflow of oil in the center, and the bottom plate is prevented from dropping out from the body by the compression spring for energizing the plunger, the surface precision or pore diameter precision as in the case of cylinder press-in is not needed for the cylindrical hole for fitting the cylinder,

and the process of surface treatment, machining or the like can be omitted. Consequently, the mounting of the cylinder and the assembling of the tensioner can be facilitated. Further, by using the compression spring for energizing the plunger, the process for the recessed groove formation in the cylindrical hole inside surface, the snap ring mounting or the like, which was performed in the past to prevent the dropping-out of the cylinder, can be dispensed with to reduce the manufacturing cost.

[0044] Further, since the metallic cylinder is only fitted to the cylindrical hole formed in the body, parts such as cylinder, plunger and the like can be used in common even if the outer shape of the body is deformed according to the mounting position on the engine, drive unit or the like so as to be conformable to various engines only by the formation of a body of new shape, and the manufacture of various hydraulic tensioners can be consequently facilitated.

[0045] In the installation of the check valve mechanism, when the cylindrical hole is provided with a cylindrical hole smaller in diameter than this cylindrical hole on the bottom side, and the check valve mechanism is fitted into the minor-diameter cylindrical hole and prevented from dropping out by the abutment of its upper part on the bottom plate of the cylinder, the prevention of dropping-out of the check valve mechanism is performed by use of the bottom plate of the cylinder, and the surface treatment, machining or the like of the minor-diameter cylindrical hole can be consequently similarly omitted to facilitate the assembling of the tensioner. Even if a looseness is caused in a temperature rise by the difference in thermal expansion coefficient in the body formed of the die cast product of aluminum, aluminum alloy or the like or the plastic product by synthetic resin molding, high-pressure oil can be supplied to the pressure oil chamber since the dropping-out of the check valve mechanism is prevented by the abutment on the bottom plate of the cylinder, and it is unnecessary to provide another measure for preventing the dropping-out of the check valve mechanism.

[0046] In the use of plastics for the body in order to reduce the weight, the insert molding of the metallic cylinder is dispensed with since the metallic cylinder is only fitted to the cylindrical hole formed in the body, and a complicated metal mold can be also dispensed with to reduce the manufacturing cost.

[0047] When the check valve mechanism comprises the check ball provided so as to block the through-hole formed in the bottom plate of the cylinder from above, the spring for energizing the check ball, and the retainer for supporting the spring, the ball seat can be abolished since the cylinder has the function of the ball seat, and the cost can be reduced.

[0048] When the ratchet pawl body to be engaged with the ratchet tooth engraved in the plunger outside surface is rockably provided on the body by spring energization in order to prevent the backward displacement of the plunger, a cutout for rocking the ratchet pawl

body engageably with the ratchet tooth and a pair of protruding pieces extending outward from the cutout end are formed on the cylinder, and the ratchet pawl body is supported by the shaft inserted to the mount holes formed in the protruding pieces and the mount hole formed in the body, the dropping-out of the cylinder from the cylindrical hole of the cylinder can be prevented since the bottom plate of the cylinder is energized downward by the compression spring provided so as to protrude the plunger tip from the body and also fixed to the body by the shaft through the protruding pieces formed on the cylinder.

[0049] Further, the shaft for supporting the ratchet pawl body is also inserted to the mount holes of the protruding pieces of the metallic cylinder, whereby the breakage of the mount hole formed in the body can be prevented in the body made of plastics, aluminum alloy or the like inferior in strength since the shaft is supported by the metallic protruding pieces.

Claims

1. A hydraulic tensioner comprising a metallic cylinder (4; 24; 34) fitted to a cylindrical hole (3; 23; 33) formed in a body (2; 22; 32), a plunger (5; 25; 35) provided within the cylinder (4; 24; 34) so as to be energized by a compression spring (7; 27; 37) to protrude the tip from the body (2; 22; 32), and a pressure oil chamber (8; 28; 38) formed between the inner part of the plunger (5; 25; 35) and the cylinder (4; 24; 34), said cylinder (4; 24; 34) being provided with a bottom plate (4B; 24B; 34B) having a through-hole (4A; 24A; 34A) allowing the inflow of oil in the center, and the bottom plate (4B; 24B; 34B) being prevented from dropping out from the body (2; 22; 32) by being energized by said compression spring (7; 27; 37).
2. A hydraulic tensioner according to claim 1 wherein said cylindrical hole (3; 23; 33) is provided with a cylindrical hole (10) smaller in diameter than this cylindrical hole (3; 23; 33) on the bottom side, and a check valve mechanism (9; 29; 39) fitted to the minor-diameter cylindrical hole (10) is prevented from dropping out by the abutment on the bottom plate (4B; 24B; 34B) of said cylinder (4; 24; 34).
3. A hydraulic tensioner comprising a metallic cylinder (4; 24; 34) fitted to a cylindrical hole (3; 23; 33) formed in a body (2; 22; 32), a plunger (5; 25; 35) provided within the cylinder (4; 24; 34) so as to be energized by a compression spring (7; 27; 37) to protrude the tip from the body (2; 22; 32), a pressure oil chamber (8; 28; 38) formed between the inner part of the plunger (5; 25; 35) and the cylinder (4; 24; 34), and a check valve mechanism (9; 29; 39) allowing the inflow of oil to the pressure oil chamber

- (8; 28; 38) but arresting the back flow thereof, wherein said cylinder (4; 24; 34) is provided with a bottom plate (4B; 24B; 34B) having a through hole (4A; 24A; 34A) allowing the inflow of oil in the center, and the bottom plate (4B; 24B; 34B) being prevented from dropping out from the body (2; 22; 32) by being energized by said compression spring (7; 27; 37); said check valve mechanism (9; 29; 39) comprises a check ball provided so as to block an inflow hole of oil to the pressure oil chamber (8; 28; 38) from above and a spring for energizing the check ball; and said spring is supported by said bottom plate (4B; 24B; 34B).
4. A hydraulic tensioner according to claim 3, wherein said check valve mechanism (9; 29; 39) further comprises a lid (9D) for supporting the spring and being a preliminary assembled product, and said lid (9D) is supported by said bottom plate (4B; 24B; 34B).
5. A hydraulic tensioner comprising a metallic cylinder (4; 24; 34) fitted to a cylindrical hole (3; 23; 33) formed in a body (2; 22; 32), a plunger (5; 25; 35) provided within the cylinder (4; 24; 34) so as to be energized by a compression spring (7; 27; 37) to protrude the tip from the body (2; 22; 32), a pressure oil chamber (8; 28; 38) formed between the inner part of the plunger (5; 25; 35) and the cylinder (4; 24; 34), and a check valve mechanism (9; 29; 39) allowing the inflow of oil to the pressure oil chamber (8; 28; 38) but arresting the back flow thereof, said cylinder (4; 24; 34) is provided with a bottom plate (4B; 24B; 34B) having a through hole (4A; 24A; 34A) allowing the inflow of oil in the center, and the bottom plate (4B; 24B; 34B) being prevented from dropping out from the body (2; 22; 32) by being energized by said compression spring (7; 27; 37) through the retainer (29A) of said check valve mechanism (9; 29; 39), and said check valve mechanism comprising (9; 29; 39) a check ball provided so as to block the through-hole (4A; 24A; 34A) formed in the bottom plate (4B; 24B; 34B) from above, a spring for energizing the check ball and a retainer (29A) for supporting the spring.
6. A hydraulic tensioner according to any one of preceding claims wherein a ratchet pawl body (40) to be engaged with a ratchet tooth (35A) engraved in the outside surface of the plunger (35) is rockably provided on said body by spring energization in order to prevent the backward displacement of the plunger (35), a cutout (34C) for rocking the ratchet pawl body (40) engageably with the ratchet tooth (34A) and a pair of protruding pieces (34D, 34D) extending outward from the end of the cutout (34C) are formed on said cylinder (34), and the ratchet pawl body (40) is supported by a shaft (42) inserted to mount holes (34E) formed in the protruding pieces (34D, 34D) and a mount hole (32A) formed in the body (32).
7. A hydraulic tensioner according to any one of preceding claims, wherein said body (2; 22; 32) is made by die-casting and said cylinder (4; 24; 34) is made of steel iron.
8. A hydraulic tensioner according to any one of preceding claims, wherein said body (2; 22; 32) is made by synthetic resin molding and said cylinder (4; 24; 34) is made of steel iron.

FIG. 1

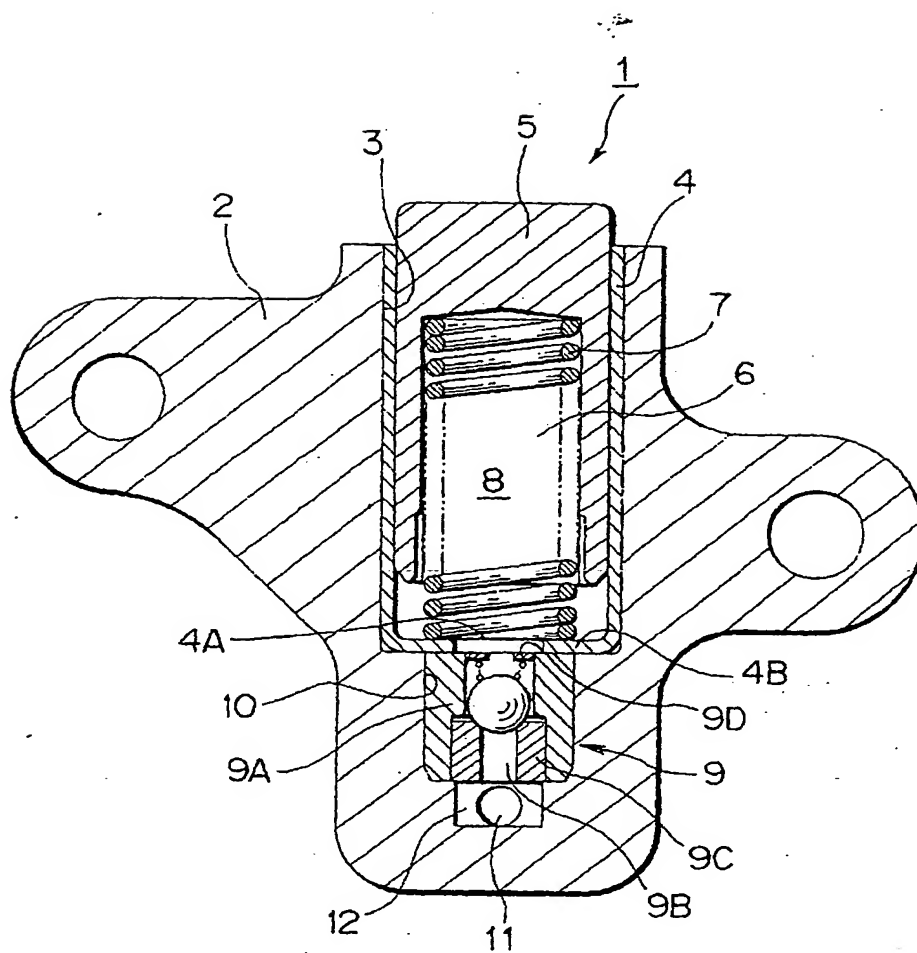


FIG. 2

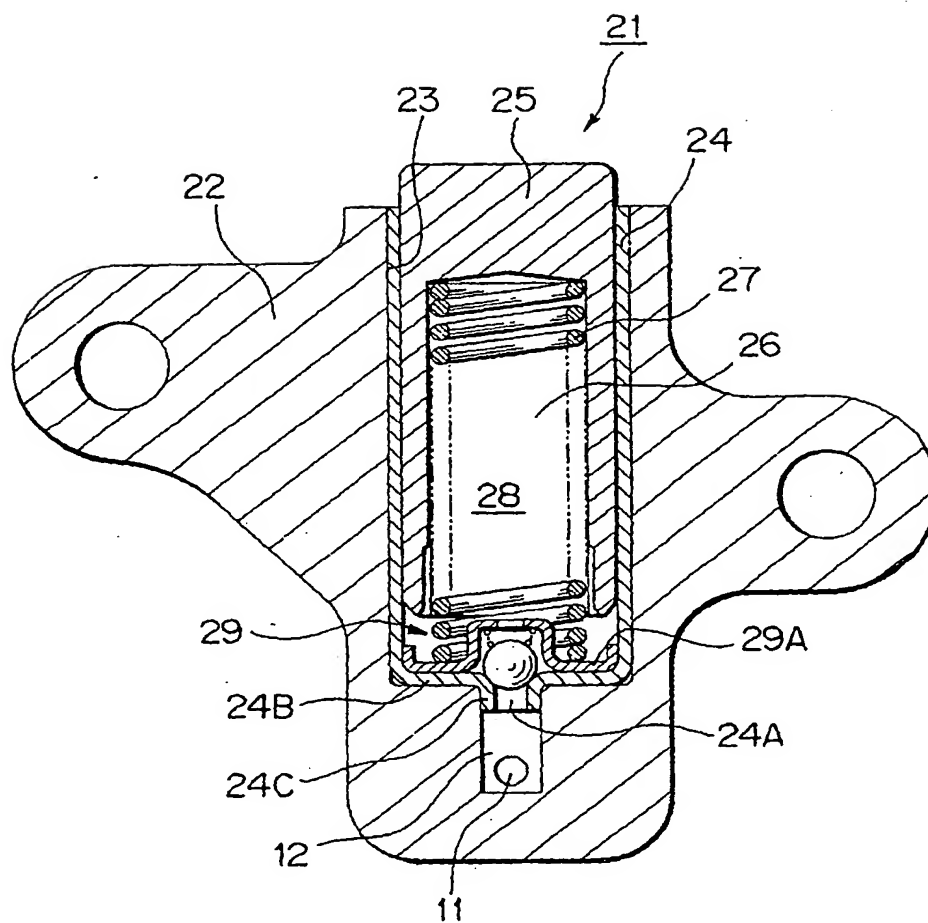


FIG. 3

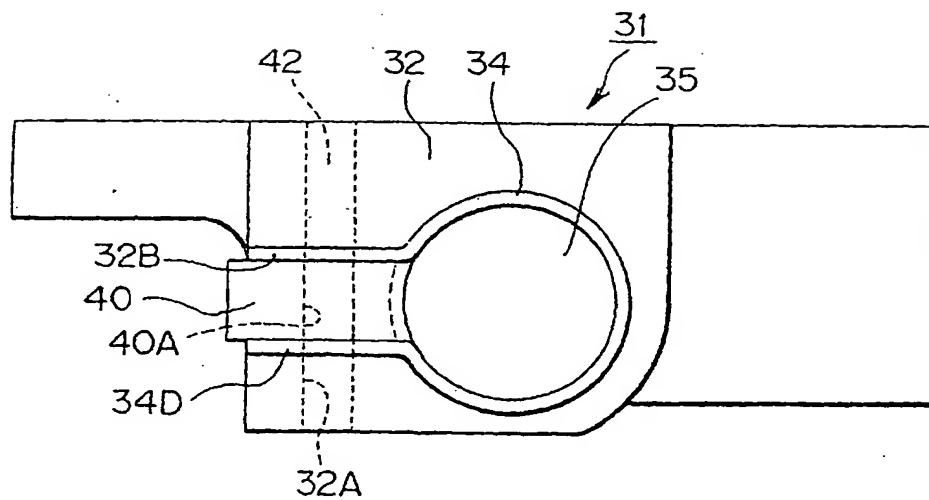


FIG. 4

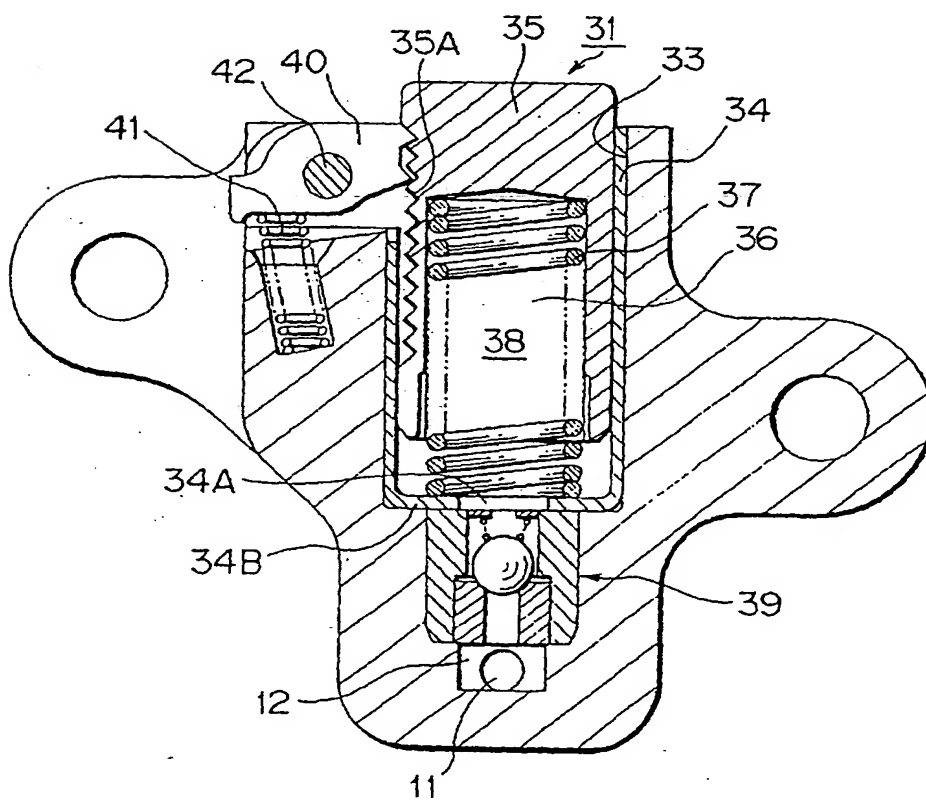


FIG. 5

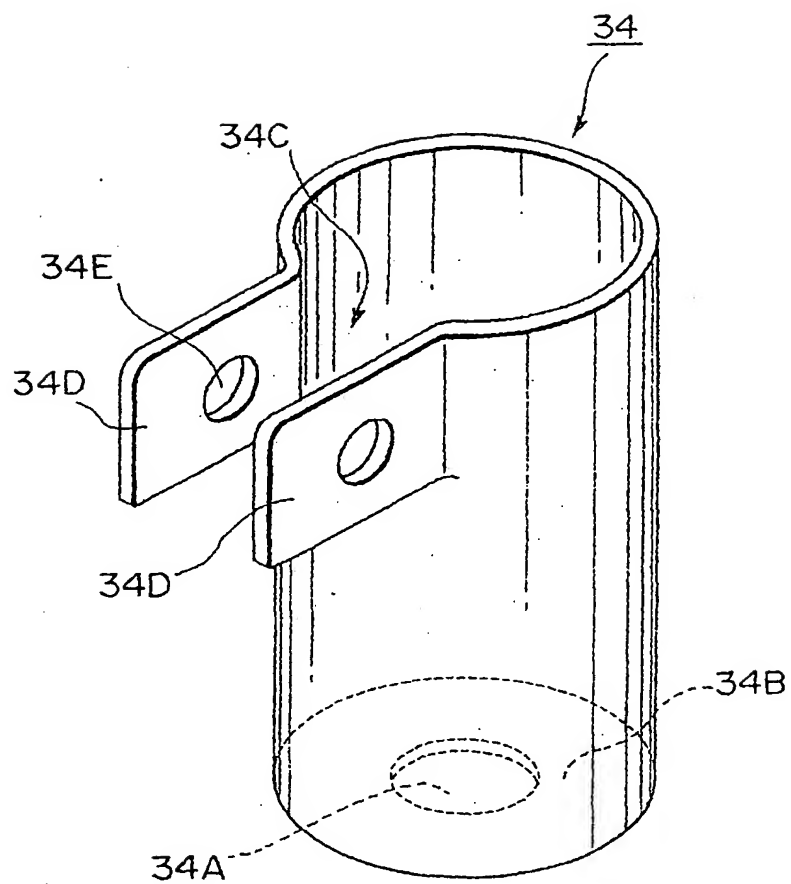
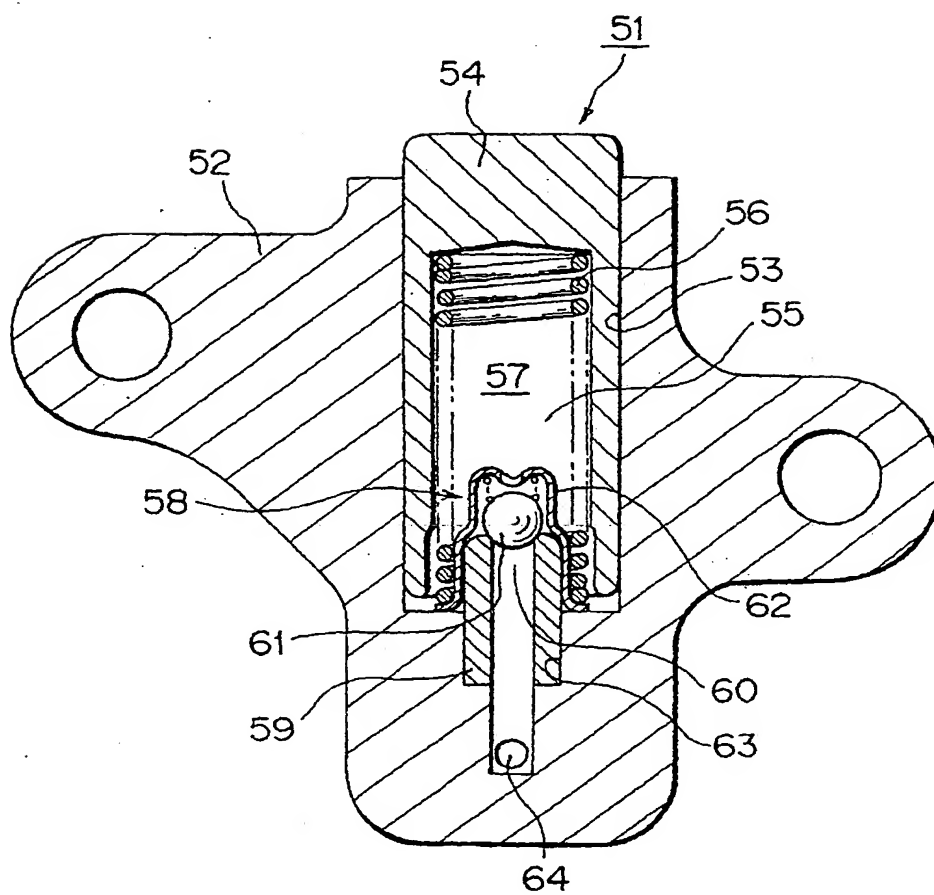


FIG. 6
PRIOR ART





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EUROPEAN SEARCH REPORT

Application Number
EP 02 00 0515

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.7)
X	EP 0 908 646 A (BORG WARNER AUTOMOTIVE) 14 April 1999 (1999-04-14) * column 5, line 14 - line 49 * * column 6, line 46 - line 48 * * column 9, line 44 - line 52 * * figures 1,2,4-6 * ---	1-5,7,8	F16H7/08
X	DE 195 38 401 A (SCHAEFFLER WAEZLAGER KG) 12 December 1996 (1996-12-12) * column 5, line 55 - column 6, line 7 * * figures 5,6 * ---	1	
Y	WO 98 05883 A (SCHAEFFLER WAEZLAGER OHG ; KOCH REINHARD (DE); ULLEIN THOMAS (DE);) 12 February 1998 (1998-02-12) * page 5, line 3 - line 9 * * figures 1-3 * ---	1,2,7	
Y	US 5 637 047 A (SCHULZE PETER) 10 June 1997 (1997-06-10) * column 4, line 36 - line 46 * * figure 2 * ---	1,2,7	
Y	EP 0 952 375 A (BORG WARNER AUTOMOTIVE) 27 October 1999 (1999-10-27) * column 7, line 7 - line 13 * * figure 5 * ---	1,8	
Y	EP 0 916 869 A (BORG WARNER AUTOMOTIVE) 19 May 1999 (1999-05-19) * abstract * * paragraph [0019] - paragraph [0021] * * figures 1,2,6,7 * ---	1,8	
-/--			
The present search report has been drawn up for all claims			TECHNICAL FIELDS SEARCHED (Int.Cl.7) F16H

Place of search BERLIN	Date of completion of the search 21 March 2002	Examiner Wilson, M
CATEGORY OF CITED DOCUMENTS X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document		



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Office

EUROPEAN SEARCH REPORT

Application Number
EP 02 00 0515

DOCUMENTS CONSIDERED TO BE RELEVANT			
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int.Cl.7)
D,A	PATENT ABSTRACTS OF JAPAN vol. 2000, no. 15, 6 April 2001 (2001-04-06) & JP 2000 346152 A (TSUBAKIMOTO CHAIN CO), 12 December 2000 (2000-12-12) * abstract *	1	
A	DE 100 07 478 A (TSUBAKIMOTO CHAIN CO) 31 August 2000 (2000-08-31) * figure 1 *	2-6	
			TECHNICAL FIELDS SEARCHED (Int.Cl.7)
The present search report has been drawn up for all claims			
Place of search BERLIN		Date of completion of the search 21 March 2002	Examiner Wilson, M
CATEGORY OF CITED DOCUMENTS X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document			

EPO FORM 1503 03.02 (P04C01)

**ANNEX TO THE EUROPEAN SEARCH REPORT
ON EUROPEAN PATENT APPLICATION NO.**

EP 02 00 0515

This annex lists the patent family members relating to the patent documents cited in the above-mentioned European search report. The members are as contained in the European Patent Office EDP file on
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21-03-2002

Patent document cited in search report		Publication date		Patent family member(s)	Publication date
EP 0908646	A	14-04-1999	US	5967920 A	19-10-1999
			EP	0908646 A1	14-04-1999
			JP	11223251 A	17-08-1999
DE 19538401	A	12-12-1996	DE	19538401 A1	12-12-1996
			DE	19680418 C1	29-11-2001
			DE	19680418 D2	20-08-1998
			WO	9641975 A1	27-12-1996
			US	5931754 A	03-08-1999
WO 9805883	A	12-02-1998	DE	19631607 A1	12-02-1998
			CN	1226953 A	25-08-1999
			DE	19780804 D2	01-04-1999
			WO	9805883 A1	12-02-1998
			US	6193623 B1	27-02-2001
US 5637047	A	10-06-1997	DE	9409155 U1	12-10-1995
			DE	59503901 D1	19-11-1998
			EP	0686787 A1	13-12-1995
EP 0952375	A	27-10-1999	EP	0952375 A2	27-10-1999
			JP	11325202 A	26-11-1999
EP 0916869	A	19-05-1999	US	5967921 A	19-10-1999
			CA	2246200 A1	09-04-1999
			EP	0916869 A2	19-05-1999
			JP	11223252 A	17-08-1999
JP 2000346152	A	12-12-2000	NONE		
DE 10007478	A	31-08-2000	JP	3243226 B2	07-01-2002
			JP	2000240744 A	05-09-2000
			DE	10007478 A1	31-08-2000

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